# Detuning Analysis and Design Optimization of an 1.3 GHz 3-cell Superconducting Cavity

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Energy recovery linac (ERL) offers a promising path to superior beam quality and substantial energy savings at high average beam currents. This paper concentrates on optimizing the structure of a 1.3 GHz 3-cell superconducting cavity for high-current ERL injectors. It particularly examines helium pressure sensitivity and the detuning mechanism in the cavity. To mitigate detuning, strategies such as optimizing stiffening rings, reducing the helium vessel diameter, and enhancing cavity stiffness are proposed. Additionally, the paper outlines the mechanical design of the cavity and evaluates its performance through tuning force, Lorentz force detuning, stress, and modal analyses to ensure robustness and reliability.

Keywords: Energy Recovery Linac, High-Current Superconducting Cavity, Detuning, Stiffening Rings, Microphonics

#### I. INTRODUCTION

The core concept of Energy Recovery Linac (ERL) is 3 that, after initial acceleration, the electron beam emits high-4 brightness radiation, while a specially designed loop guides 5 it back to the accelerating section in a decelerated state. This 6 process recovers the electron beam's kinetic energy, convert-7 ing it into high-frequency electromagnetic waves, which are 8 then used to accelerate new electron beams, while the de-9 celerated old beam is discarded. This design significantly 10 reduces energy consumption and ensures the quality of the 11 electron beam, making ERL an effective method for generat-12 ing high quality electron beams[1]. ERL projects worldwide 13 have demonstrated remarkable advancements in beam energy 14 recovery and operational efficiency. The CEBAF experiment achieved a maximum energy of 1055 MeV in single-pass con-16 figurations with a continuous-wave (CW) beam current of  $_{17}$  80  $\mu$ A[2, 3]. The CBETA project demonstrated multi-turn en-18 ergy recovery using superconducting radio-frequency (SRF) 19 cavities and a fixed-field alternating-gradient (FFAG) system. 20 The permanent magnets are arranged in the FFAG system construct a single return loop that successfully transports 22 electron bunches of 42, 78, 114, and 150 MeV in one com-23 mon vacuum chamber[4, 5]. The cERL facility at KEK has 24 successfully demonstrated stable CW operation at 1 mA with beam energies ranging from 17.6 to 20 MeV, achieving en-26 ergy recovery efficiency exceeding 99.9%[6, 7]. The recircu-27 lating superconducting linear accelerator S-DALINAC at TU 28 Darmstadt's Institute for Nuclear Physics is a key research tool. It operates in CW mode, achieving beam currents up to  $_{30}$  20  $\mu$ A and energies as high as 130 MeV, employing a threepass recirculation scheme[8]. Future projects such as MESA 32 at Mainz University in Germany, PERLE at IJC in France, the

33 five-pass CEBAF at JLAB, and the EIC Cooler at Brookhaven 34 National Laboratory in the U.S. will all rely on high-current SRF accelerator technologies due to their ability to achieve high beam intensities, efficient energy recovery, and stable continuous-wave operation[9–12]. Together, these projects 38 underscore the transformative potential of ERL technology in 39 achieving high-energy efficiency and beam quality. Based on 40 the recent developments of the continuous-wave SRF accel-41 erator technology[13–19], the project of this paper also inves-42 tigates ERL-based linear accelerators and proposes a concept  $_{43}$  to achieve an average operational beam current of  $10\,\mathrm{mA}.$ 44 This study presents an in-depth analysis and optimization of 45 the mechanical structure of an 1.3 GHz high-current injector 46 SRF cavity, significantly enhancing its frequency stability and 47 mechanical performance. Additionally, a comprehensive sim-48 ulation methodology for evaluating the stability and strength 49 of SRF cavities has been developed. This methodology es-50 tablishes a solid theoretical foundation for future cavity man-51 ufacturing and tuning processes, providing valuable engineer-52 ing insights for research.

To meet the 10 mA average beam current requirement of 54 the ERL project, a complete radio-frequency (RF) param-55 eter design of the injector cavity was implemented and an 1.3 GHz 3-cell injector superconducting cavity configuration was selected[20, 21]. The vacuum model of the SRF cavity is shown in Fig. 1. This study focuses on the design and performance evaluation of an 1.3 GHz 3-cell injector superconducting cavity for an energy recovery linac aimed at achieving an average current of 10 mA. Based on the physical design, the 62 mechanical structure of the high-current 1.3 GHz SRF cavity was designed and evaluated, with emphasis on key parameters such as mechanical strength, helium pressure detuning, Lorentz detuning, and modal analysis. A multiphysics opti-66 mization approach was applied to the cavity, with particular 67 attention given to detuning and stability. Finite element simulation software was used to optimize and reinforce the cavity 69 structure which is crucial for the fabrication of high-current 70 SRF cavities. Through design optimization, the cavity's fre-71 quency and phase deviations caused by small deformations 72 during operation were effectively controlled. This research

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74 SRF cavities and offers important theoretical support for the 114 mizes the cavity's mechanical design to increase its natural 75 fabrication process.

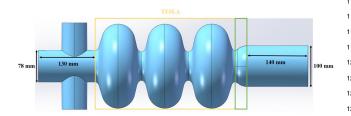


Fig. 1. Vacuum model of the 1.3 GHz 3-cell SRF injector cavity. The green area represents the designed transition section. The yellow area maintains the same mid-cell and end-cell dimensions as the TESLA cavity.

Chapter II details the theoretical framework of key de-77 tuning mechanisms in superconducting cavities, establish-78 ing a foundation for subsequent analysis. Chapter III con-79 ducts a systematic investigation of helium pressure sensitivity 80 (df/dp) across different tuner stiffness configurations, with 81 the study specifically aimed at quantifying the cavity's me-82 chanical response to pressure fluctuations. This analysis uti-83 lizes multiphysics-coupled finite element simulations to cor-84 relate structural deformations with frequency detuning, providing critical insights into the cavity's stability. Chapter IV 86 systematically formulates the primary factors contributing to 87 cavity detuning, quantifies their individual effects, and validates the results through comparisons with full-model simulations. Chapter V focuses on the mechanical design of the cavity, covering the evaluation of tuning forces and Lorentz force detuning, and concludes with a thorough assessment of 92 the cavity's structural strength to ensure its operational relia-93 bility.

# DETUNING THEORY OF SUPERCONDUCTING

### A. Microphonics

Superconducting cavities are typically designed with thin walls, usually less than 3 mm[22], to meet heat dissipation and tuning requirements. This design makes them highly susceptible to deformation from external disturbances. Mechanical resonance occurs when the frequency of external vibrations closely matches the natural mechanical resonance frequency of the cavity. This phenomenon, termed microphonics, leads to substantial structural deformation and frequency 105 instability. External sources of vibration, including ambient noise, equipment operation, and structural vibrations, are the 155 primary causes of this effect.

109 frequency superconducting cavities and prevent resonance, 158 the RF field and the cavity material induces small deformathree strategies are commonly used: the first involves adding 159 tions, particularly in the superconducting material (e.g., nio-111 vibration isolation systems to reduce the transmission of en- 160 bium). As the field strength increases, elastic deformation 112 vironmental vibrations, the second utilizes low-level control 161 alters the geometry of the cavity, changing its RF resonant

73 provides valuable insights into the mechanical properties of 113 compensation to counteract vibrations, and the third optimechanical resonance frequency, reducing its susceptibility 116 to resonance[23]. The first two methods focus on optimizing the external environment and active control. And this paper focuses on the third approach, improving the cavity's structural design to improve its natural machanical resonance fre-

> During the design phase, analyzing the mechanical vibration modes of the cavity helps to identify its natural mechanical frequencies. External vibrations are typically below 50 Hz and have relatively large amplitudes. As a result, the lowfrequency modes within the superconducting cavity are more 126 likely to resonate with environmental vibration sources. In-127 creasing the cavity's mechanical resonance frequency minimizes the risk of resonance with low-frequency environmen-129 tal vibrations, reducing microphone detuning effects.

#### B. Helium pressure fluctuation

In superconducting accelerator systems, the liquid helium 132 tank plays a crucial role in maintaining the low-temperature operating state of the superconducting cavity. Fluctuations 134 and pressure changes in the liquid helium not only affect the 135 cooling performance of the cavity but can also have a signif-136 icant impact on its frequency stability. In particular, helium 137 pressure fluctuations within the tank, especially in cryogenic 138 environments, can lead to detuning of the cavity's frequency. 139 This phenomenon is known as helium pressure fluctuation de-

Liquid helium fluctuations cause pressure variations within 142 the helium tank, leading to surface deformation of the cavity, 143 which in turn results in changes to the cavity's RF resonant 144 frequency[24]. This deformation is directly proportional to 145 the helium pressure fluctuations. The mathematical relation-146 ship between the cavity frequency and the helium pressure variation can be expressed as [25]:

$$\Delta f_1 = \frac{df}{dp} \times \Delta P \tag{1}$$

Here df/dp denotes the helium pressure sensitivity coefficient, which characterizes the sensitivity of the superconducting cavity to fluctuations in the helium bath pressure within 152 the cryostat.  $\Delta P$  represents the magnitude of the fluctuations 153 in helium pressure.

#### C. Lorentz force detuning

Lorentz force detuning (LFD) arises from the inelastic de-156 formation of the superconducting cavity material under the To mitigate the impact of external vibrations on high- 157 influence of a strong RF field[26]. The interaction between

162 frequency. The effect of the Lorentz force on an elliptical 185 163 cavity is shown in Fig. 2[27]. When the accelerating gradient 186 more significant impact, while helium pressure fluctuations 164 is 4.77 MV/m, the Lorentz force distribution calculated by 187 are less influential [29, 30]. In contrast, for high-current ERL 165 simulation is shown in the Fig. 3.

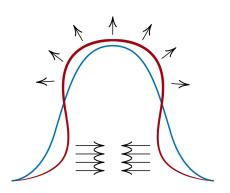


Fig. 2. Schematic of Lorentz force in an elliptical cavity. The blue contour denotes the original cell shape before deformation, the red blue contour denotes the deformed shape due to LFD.

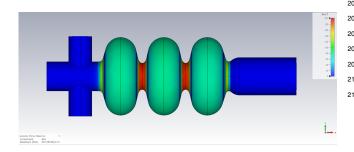


Fig. 3. Lorentz force distribution of the 3-cell superconducting cavity. When the accelerating gradient is 4.77 MV/m, the Lorentz force is highest at the cavity iris, with a peak value of  $201 \,\mathrm{N/m^2}$ .

The Lorentz force is generated by the electromagnetic field 169 within the cavity and acts on the inner surface of the cavity, 170 leading to deformation-induced detuning. The magnitude of the Lorentz force (P) is given by Equation[28]:

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$$P = \frac{1}{4}(\mu_0 H^2 - \epsilon_0 E^2)$$
 (2)

173 Here  $\mu_0$  is the vaccum permeability,  $\epsilon_0$  is the vaccum per- $^{174}$  mittivity, and E and H are the magnetic field strength and electric field strength at the inner surface of the cavity, re-

Under different values of accelerating gradient  $E_{acc}$ , the de-177 178 formation of the cavity and the corresponding frequency shift due to the Lorentz force vary. The frequency shift  $\Delta f$  is proportional to the square of the accelerating gradient  $E_{\rm acc}^2$ :

$$\Delta f_2 = K_L \times E_{\rm acc}^2 \tag{3}$$

frequency effect of the Lorentz force detuning on the super- 223 and  $\epsilon_0$  set at  $8.854 \times 10^{-12}$  F/m. For the mechanical struc-184 conducting cavity.

For superconducting cavities in pulsed mode, LFD has a 188 cavities in CW mode, Lorentz detuning primarily affects the 189 early field buildup, so the focus is on mitigating the effect 190 of helium pressure fluctuations, with Lorentz detuning as a 191 secondary concern[31].

#### III. THE COMPLETE MODEL DETUNING ANALYSIS

Based on the internal profile parameters of the designed superconducting cavity[20], the cavity's outline can be sketched and modeled in CAD software. To minimize potential errors during the modeling process, the vacuum model established in the electromagnetic simulation phase is imported directly. Using the shell feature, the cavity's outer shell is automatically generated. The outer shell model is then precisely assembled with the vacuum model, effectively preventing interference issues in subsequent finite element analysis and ensuring the accuracy of the simulation.

The typical thickness for an 1.3 GHz superconducting cav-204 ity ranges from 2.5 to 3 mm[32]. Insufficient thickness compromises the structural strength of the cavity, while excessive thickness could lead to challenges in welding and material waste. The thickness of the 1.3 GHz 3-cell high-current superconducting cavity is set to 2.8 mm, which is similar to the TESLA cavity design[33]. The model of the dressed SRF 210 cavity is shown in Fig. 4. The helium tank design and de-211 tailed parameters will be discussed in Chapter V.

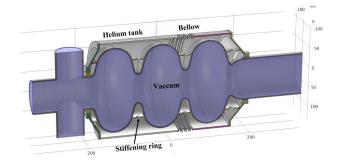


Fig. 4. Mechanical model (grey color) of 3-cell dressed SRF cavity with vacuum part (purple color) inside.

The elliptical cavity's relatively weak stiffness is one of its 213 disadvantages. Therefore, an analysis of the detuning caused 214 by helium pressure fluctuations, mechanical vibrations, and 215 Lorentz forces is a necessary step in the development of the 216 elliptical cavity. The accuracy of simulations largely depends 217 on the precision of material properties. Given that the su-218 perconducting cavity involves multi-physics coupling of elec-219 tromagnetic, mechanical, and thermal fields, various material (3) 220 properties must be considered. In electromagnetic field sim-221 ulations, two primary materials are involved: Perfect electric Here  $K_L$  is Lorentz detuning coefficient. It represents the 222 conductor (PEC) and vacuum, with  $\mu_0$  set at  $4 \times 10^{-7}$  H/m 224 tural simulations, the mechanical properties of materials such

225 as niobium, niobium-titanium alloy, and titanium for the he-  $^{252}$   $10\,\mathrm{kN/mm}$ ,  $40\,\mathrm{kN/mm}$ , and  $80\,\mathrm{kN/mm}$ . The simulation re-226 lium tank are required. The relevant mechanical properties of 253 sults are shown in Fig. 6. The outer diameter of the bare cav-227 these materials, which can influence the cavity strength and 254 ity iris is 38 mm, and the outer diameter of the equator is 105 228 stability, are summarized in Table 1[34, 35].

Table 1. Properties of Superconducting Cavity Materials.

Performance parameters	Nb	Ti	Nb55Ti
Density $[kg/m^3]$	8570	4506	5700
Young's Modulus $[GPa]$	118	117	68
Poisson's Ratio	0.38	0.37	0.33
Allowable stress (S) in $4 \text{ K } [MPa]$	212.0	319.0	156.0
Allowable stress (S) in $293 \mathrm{K} \; [MPa]$	47.0	98.0	156.0

The helium pressure detuning (df/dp) of a 3-cell highcurrent superconducting cavity was simulated using COM-SOL Multiphysics simulation software[36]. One end of the cavity's beam tube was fixed, while the other end was allowed to expand freely. A pressure of 1 bar was applied to both the external surface of the bare cavity and the inner surface of the helium tank to simulate pressure fluctuations within 259 236 the dressed cavity. Meanwhile, in finite element analysis, 260 parts with complex structures that have minimal impact on the simulation results should be appropriately optimized to save 239 computational resources. For instance, the bellows with an intricate and convoluted design require a high mesh density to achieve its stiffness accurately. To optimize computational 242 efficiency, this study employs a material stiffness parame-243 ter equivalence approach, replacing the actual bellows model 244 with an equivalent straight pipe of the same length. This 245 method preserves the original stiffness characteristics while 246 significantly reducing mesh resource consumption. The sim-247 ulation boundary condition and the equivalent straight pipe 248 model are shown in Fig. 5.

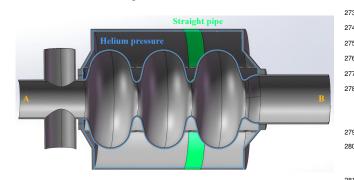


Fig. 5. A and B represent the beam tubes at both ends, helium presequivalent straight pipe model of the bellows.

The df/dp was evaluated as a function of the stiffening 250 ring radius through multiphysics finite element simulations, with the analysis spanning three tuner stiffness conditions: 287 Here  $\Delta f_{\rm cell}$  is the frequency shift caused by the shape vari-

255 mm, so the range of possible stiffening ring radii is between  $_{256}$  38 mm and 105 mm, with  $r_{ring} = 38$  mm representing the 257 case with no stiffening ring.

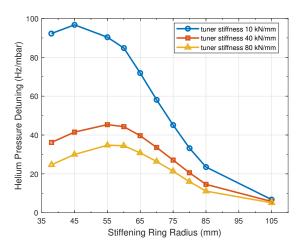


Fig. 6. df/dp of the 3-cell cavity for stiffening ring radii from 38 to  $105 \,\mathrm{mm}$  and tuner stiffnesses from 10 to  $80 \,\mathrm{kN/mm}$ .

The helium pressure sensitivity initially increases with the 261 stiffening ring radius and then begins to decrease signifi-262 cantly. In the TESLA cavity, the stiffening ring is located  $_{263}$  at the region where df/dp reaches its approximate maximum <sup>264</sup> value. This is because the primary optimization goal for the 265 stiffening ring in the TESLA cavity is to minimize Lorentz 266 detuning rather than the microphone effect[37]. In contrast, when the stiffening ring is positioned at the equator, df/dp268 reaches its minimum. For a tuner stiffness of 40 kN/mm[38], 269 the optimal stiffening ring configuration to minimize df/dp270 was identified at the maximum radius position, followed by 271 the configuration with no stiffening ring. For all stiffening 272 ring radii, increasing the tuner stiffness results in a reduction of df/dp. This simulation provides a good assessment of the  $_{274}$  df/dp for the entire cavity, but it does not directly explain the variation of df/dp with stiffening ring radius and tuner stiffness, nor does it account for the influence of other specific components. Next, the factors influencing df/dp were analyzed and simulated individually.

#### DECONSTRUCTIVE ANALYSIS AND VALIDATION OF HELIUM PRESSURE DETUNING

To investigate the factors contributing to the variation of  $_{282}$  df/dp in detail, the total frequency variation of the cavity sure is inside helium vessel, and the green section represents the 283 is divided into two components: the frequency shift caused 284 by changes in cavity length and the frequency shift resulting 285 from changes in the shape of the cavity cells:

$$\Delta f_{total} = \Delta f_{\text{cell}} + \Delta f_{\text{length}} \tag{4}$$

<sup>288</sup> ations of the cavity cells, and  $\Delta f_{ ext{length}}$  is the frequency shift <sup>320</sup> 289 caused by the cavity length change. The above equation di- 321 has a impact on df/dp, fluctuating between  $-5\,\mathrm{Hz/mabr}$  and 290 vided by the change in helium pressure[39]:

$$\frac{df}{dp} = \frac{df_{\text{cell}}}{dp} + \frac{df_{\text{length}}}{dp} \tag{5}$$

The frequency shift  $\Delta f_{\text{cell}}$  results from the direct effect of 292 293 liquid helium pressure on the outer wall of the cells. The helium bath pressure causes deformation of the cell walls. 294

The frequency shift  $\Delta f_{\text{length}}$  due to the cavity length change 296 is attributed to the liquid helium's effect on the conical discs at both ends of the helium tank. The pressure from the he-298 lium bath exerts a stretching force at the ends, which leads to a change in the overall cavity length, thereby affecting the 300 tuning of the cavity. The length variation can further be decomposed into two components: the original cavity length xand the force F applied to the cavity due to the elongation.

$$\frac{df_{\text{length}}}{dp} = \frac{df}{dx} \cdot \frac{dx}{dF} \cdot \frac{dF}{dp} = \frac{df}{dx} \cdot \frac{1}{K} \cdot \frac{dF}{dp}$$
 (6)

 $_{304}$  Here  $df_{length}/dx$  represents the tuning sensitivity of the bare cavity, and K denotes the axial stiffness of the dressed cavity.

These parameters will be simulated and analyzed to eval-306 307 uate the specific factors influencing df/dp. The simulation 308 conditions for each parameter are listed in Table 2, and the 309 corresponding loads applied can be seen in 5.

#### Helium pressure sensitivity of the bare cavity

First, the helium pressure detuning component  $\Delta f_{\text{cell}}$  is an-312 alyzed. In COMSOL, both ends of the beam pipe are fixed, and a pressure of 1 bar is applied to simulate the helium pressure acting solely on the outer wall of the bare cavity, to investigate the helium pressure detuning caused by the deformation of the cavity wall, denoted as  $df_{\text{cell}}/dp$ . The results 317 are shown in Fig. 7.

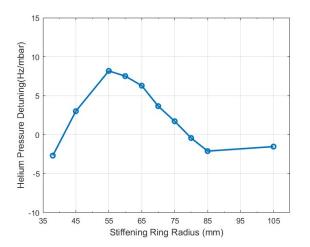


Fig. 7.  $df_{cell}/dp$  of cavity versus stiffening rings radius.

The results indicate that the variation in the bare cavity wall 322 10 Hz/mbar. In the configuration without a stiffening ring, 323 the  $df_{
m cell}/dp$  value is lower compared to the configuration 324 with the stiffening ring placed at a radius of approximately 325 55 mm. This is because, as the helium pressure increases, the 326 negative frequency shift caused by the deformation of the iris 327 counteracts the positive frequency shift caused by the defor-328 mation of the equator and the increase in length.

#### B. The tuning sensitivity of the bare cavity

The tuning sensitivity  $df_{length}/dx$  of the bare cavity is in-331 fluenced by the cavity shape and dimensions. Using COM-SOL simulations, the relationship between the tuning sensi-333 tivity  $df_{length}/dx$  of the 3-cell cavity and the stiffening ring <sup>334</sup> radius is simulated and presented in Fig. 8.

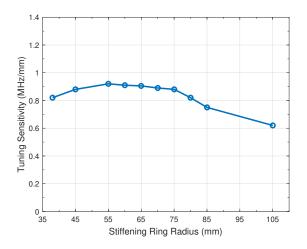


Fig. 8. Tuning sensitivity of cavity versus stiffening rings radius.

The results indicate that  $df_{length}/dx$  of the 3-cell su-338 perconducting cavity varies fractionally as the stiffening  $_{339}$  ring radius changes, fluctuating between  $0.7\,\mathrm{MHz/mm}$  and 1 MHz/mm. Overall, the stiffening ring radius does have some effect on  $df_{length}/dx$ , but this influence is relatively 342 small.

#### Axial stiffness and helium pressure tuning force

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The cavity axial stiffness is an essential factor in the study of helium pressure detuning. Stiffness simulations were conducted for both the bare cavity and the dressed cavity, with the comparative results presented in Fig. 9.

In addition to exerting pressure on the cavity walls, the liquid helium in the helium bath also induces deformation of the bath wall itself due to the pressure, thereby generating a 352 tensile force in the axial direction on the cavity. For the 3-353 cell cavity, the pressure on the end wall of the helium bath has the most significant impact on the dF/dp. As shown in

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Table 2. Simulations for determining df/dp model parameters.

Parameter	Boundary conditions	Load	Calculation results
$d\!f_{ m cell}/dp$	A, B tubes fixed	1 bar helium pressure	frequency variation
$df_{length}/dx$	A tube fixed	1 mm displacement (B)	frequency variation
K	A tube fixed	2.5 kN axial force (B)	Length change
dF/dp	A, B tubes fixed	1 bar helium pressure	Average force

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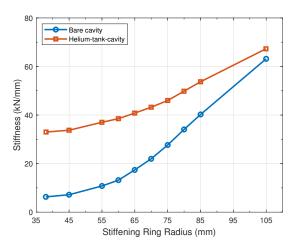


Fig. 9. Stiffness of bare cavity and dressed cavity versus stiffening rings radius.

355 Fig. 10, for a cavity with a stiffening ring radius of 70 mm as an example, the pressure applied on the end wall of the he-357 lium bath, which has a small outer diameter of 120 mm, was 358 simulated and integrated in COMSOL to calculate the result-359 ing force. When the internal pressure was 1 bar, the integral 360 result of the force component in the x-direction (axial direc- $_{361}$  tion),  $dp_{\rm x}$ , was  $3499.4\,{\rm N}$ . Therefore, for the 3-cell cavity,  $_{362}\ dF/dp = 3499.4\ \text{N/bar}.$ 

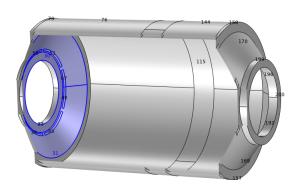


Fig. 10. Integral region of pressure on the helium tank end wall.

366 end wall of the helium bath on df/dp, a cavity model with an end wall outer diameter expanded to 150 mm was estab-368 lished, while keeping other parameters unchanged. The in- $_{369}$  tegral result of the force component  $dp_{\rm x}$  was  $6023.3\,{\rm N}$ . The detuning simulation results are shown in Table 3. It can be 371 observed that as the outer diameter of the helium bath's end wall increases, the df/dp of the cavity rises rapidly.

Table 3. Simulation comparison of dressed cavities with different outer diameters for stiffening rings radius of 70 mm.

Detuning	$120\mathrm{mm}$	$150\mathrm{mm}$
$df/dp$ @ $K_{\text{tuner}}10\mathrm{kN/mm}$	58.07	114.73
$df/dp$ @ $K_{\rm tuner}40{\rm kN/mm}$	33.49	95.10
$df/dp$ @ $K_{\rm tuner}$ 80 kN/mm	26.32	90.79

The main source of the cavity length change caused by 376 helium pressure is the pressure exerted by the helium bath on the end walls of the helium tank. When using the same 378 type of tuner, the ratio of dF/dp for different cavities is ap-379 proximately equal to the ratio of the endwall areas of the 380 helium bath. Therefore, reducing the diameter of the he-381 lium bath is a feasible approach to optimize helium pressure 382 detuning[40, 41].

# D. Summary of helium pressure sensitivity model

Building on the previous simulations, an independent analysis was conducted for each influencing factor outlined in Eq. (6), and the impact of each factor was quantified through simulation. Fig. 11 shows the results of  $df_{length}/dp$  under the condition of a tuner stiffness of 40 kN/mm. Furthermore, the influence of cell cavity wall deformation,  $df_{\rm cell}/dp$ , is also plotted. The sum of these two components is compared with the complete simulation result of df/dp for the entire cavity[39].

The decomposition prediction of the model is in close agreement with the complete simulation of the entire cavity. The difference between the two is less than 5% at most of the data points, validating the reliability of the helium pressure detuning model and its conclusions. Therefore, increasing K, or reducing  $df_{\text{cell}}/dp$ , tuning sensitivity  $k_{\text{tune}}$ , or helium tank end-wall area S, are all effective methods to reduce df/dp.

For the 3-cell cavity without a stiffening ring, the impact of To further investigate the effect of helium pressure on the 402 cavity wall deformation on frequency is relatively small. This

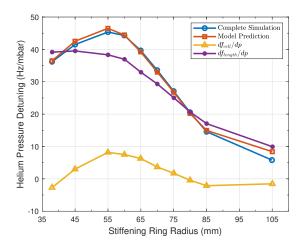


Fig. 11. Comparison of model combinations and complete cavity simulations.  $df_{cell}/dp$  is the frequency shift caused by the shape variations of the cavity cells, and  $df_{length}/dp$  is the frequency shift caused by the cavity length change. The combined model result is derived by formulating the individual simulation results from above, and it closely matches the curve obtained from the completed dressed cavity simulation.

403 is because the positive frequency shift caused by the equatorial deformation counteracts the negative frequency shift caused by the iris deformation [42]. For the 3-cell cavity with the stiffening ring, the stiffening ring changes the balance. And the stiffening ring changes the stiffness  ${\cal K}$  and limits the deformation of the bare cavity, thereby affecting  $df_{\text{length}}/dp$ 408 and  $df_{\text{cell}}/dp$ . 409

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When selecting the location of the stiffening ring, multiple 411 factors need to be considered. The cavity's strength cannot be 412 too weak, so a stiffening ring is required to ensure its struc-413 tural integrity. If the stiffening ring radius of the cavity is set in the region where df/dp is relatively low, meaning the 415 stiffening ring radius is larger, this effectively reduces the de-416 tuning caused by helium pressure variations. However, it also 417 results in greater tuning difficulty. Therefore, after balancing 418 the reduction of df/dp with tuning flexibility, this study pre- $^{419}$  liminarily sets the stiffening ring radius at  $70 \,\mathrm{mm}$  to optimize 420 the performance of the superconducting cavity.

#### MECHANICAL STRUCTURE ANALYSIS

Building upon the findings from the previous chapter, the 422 423 fundamental structural parameters of the superconducting cavity are further refined. The thickness of the bare cavity wall is set at 2.8 mm, and the stiffening ring radius is established at 70 mm to enhance the structural stability. Additionally, to ensure the deformation consistency between the end cell and the middle cell during tuning, a stiffening ring is 451 also installed at the position of the U-shaped beam pipe. The 452 430 thickness of the stiffening ring is set to 3 mm, and to main- 453 431 tain helium flow, six evenly distributed holes are placed on 454 the 3-cell superconducting cavity as a function of the stiff-492 the stiffening ring. The hole diameter between the cells is set 455 ening ring radius is shown in Fig. 13. The results indicate

 $_{433}$  to  $10\,\mathrm{mm}$ , while the holes connecting the end cells and the 434 U-shaped beam pipe are larger, with a diameter of 16 mm. 435 Fig. 12 shows the sectional view of the bare cavity.

This study also includes an in-depth investigation of the mechanical performance of the 1.3 GHz superconducting cavity, covering the simulation of the tuning force and microphonics, the analysis of the Lorentz force detuning and the evaluation of the pressure tolerance.



Fig. 12. Cross-sectional view of the bare cavity with a 70 mm stiffening ring radius, highlighting the structural configuration.

# The analysis of the tuning force

In the cryomodule, the frequency fine-tuning of the super-443 conducting cavity is achieved through a precise piezo tuner. 444 However, before the cavity is assembled into the cryomodule, 445 coarse frequency tuning must be performed using a mechani-446 cal tuner. For an 1.3 GHz superconducting cavity, the typical 447 tuning range is 500 kHz[43]. To ensure that the cavity is not 448 damaged within the tuning range, it is necessary to simulate 449 the force required for a 500 kHz frequency shift during the 450 tuning process.

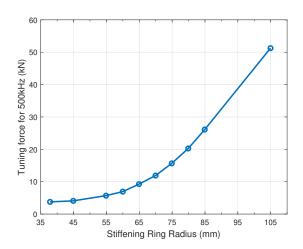


Fig. 13. Tuning force for 500 kHz versus stiffening rings radius.

The force required to tune a frequency shift of 500 kHz in

456 that as the radius of the stiffening ring increases, the re-457 quired tuning force rises exponentially. When the radius ex-458 ceeds 85 mm, the tuning force becomes very large (greater 459 than 30 kN), which poses significant challenges for the tuner 460 design. Therefore, the cavity with a 70 mm stiffening ring 461 radius demonstrated compliance with operational specifica-462 tions, while the required tuning force of less than 15 kN significantly reduced mechanical adjustment complexity and 464 simplified the tuner design requirement.

#### The analysis of microphonics

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The microphone effect is closely related to the mechani-466 467 cal vibration modes of the superconducting cavity. When the 468 cavity's natural mechanical modes are influenced by exter-469 nal vibrations, it can trigger mechanical vibrations within the 470 superconducting cavity, sometimes causing detuning of the RF resonant frequency. The structure of the superconducting cavity has a specific response pattern to external disturbances, with mechanical vibrations being transmitted to the cavity through transmission lines, low-temperature systems, 497 475 supports, and the ground. When the frequency of the exter-476 nal vibration matches the inherent mechanical resonance fre-477 quency of the cavity, mechanical resonance can occur, lead- $_{478}$  ing to deformation of the cavity and subsequently affecting  $_{501}$  MV/m and a tuner stiffness of  $40\,\mathrm{kN/mm}$ , the frequency shift 479 the stability of the RF resonant frequency. To ensure the stable operation of the superconducting cavity and prevent lowfrequency resonance caused by environmental vibrations, it 481 is crucial to understand and enhance the intrinsic vibration 482 frequency of the cavity.

1.3 GHz superconducting cavity with stiffening rings radius of 70 mm, and the simulation results are shown in Fig. 14. Additionally, the modal analysis results for the cavity without the stiffening ring and the cavity with 70 mm stiffening ring are presented in Table 4. The results indicate that even without the stiffening ring, the lowest modal frequency of the cav-491 ity reaches 199.58 Hz, which is significantly higher than the 492 typical vibration frequency of 100 Hz commonly observed in 493 standard equipment[44]. This demonstrates that the cavity's 498 vibration modes meet the design requirements.

Table 4. Mechanical modes of 3-cell superconducting cavity.

	w/o stiffening ring	$70\mathrm{mm}$ stiffening ring
Mode 1 [Hz]	199.58	266.43
Mode 2 [Hz]	205.66	299.36
Mode 3 [Hz]	339.88	449.16
Mode 4 [Hz]	357.76	513.51
Mode 5 [Hz]	403.96	619.58
Mode 6 [Hz]	478.29	735.89

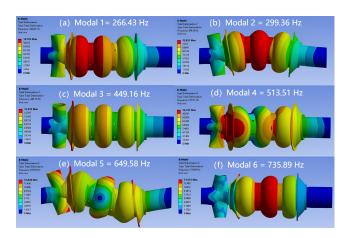


Fig. 14. The lowest six mechanical modes of the cavity with stiffening rings. For the cavity with a 70 mm radius and a stiffening ring, the lowest natural mechanical frequencies of (a) Mode 1 and (b) Mode 2 are significantly higher than 50 Hz.

#### C. The analysis of Lorentz force detuning

In the CW operation of the ERL high-current supercon-498 ducting cavity, Lorentz detuning primarily affects the cavity 499 during the initial field buildup phase. Simulation results are 500 shown in Fig. 15, under the design accelerating gradient of 12 502 caused by Lorentz forces ranges between 138 Hz and 197 Hz. 503 The 70 mm stiffening ring radius is optimized for minimizing df/dp, and at this point, the superconducting cavity exhibits a 505 relatively high Lorentz detuning coefficient, with a detuning 506 amount of 162 Hz. Despite the relatively large detuning, this Based on this, a modal analysis was performed on an 507 does not imply that the cavity cannot operate normally. By 508 employing feedforward control, the effects of Lorentz detuning can be effectively compensated and eliminated [45, 46].

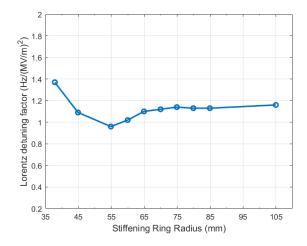


Fig. 15. Lorentz detuning factor of cavity versus stiffening rings radius.

Feedforward control is a technique that preemptively compensates for detuning before it occurs. The feedforward con-

Condition	Loads	Boundary	Temperature	Peak Stress on Cavity	Allowable Stress
Leak check (bare cavity)	i) Gravity   ii) $P_1=0$ iii) $P_2=P_3=0.1$ MPa	Both ends fixed	293 K	5.30	47.0 (S) 70.5 (1.5S)
Vertical test (bare cavity)	i) Gravity $ \label{eq:power_problem} \mbox{ii) } P_1 = 0 \\ \mbox{iii) } P_2 = P_3 = 0.15 \mbox{ MPa} $	Both ends fixed	$4.2\mathrm{K}$	7.95	212 (S) 318 (1.5S)
Horizontal test (dressed cavity)	i) Gravity ii) $P_1 = P_3 = 0$ iii) $P_2 = 0.2$ MPa	Single-end fixed	$4.2\mathrm{K}$	18.4	212 (S) 318 (1.5S)
Extreme situation (dressed cavity)	i) Gravity ii) $P_1 = P_3 = 0$ iii) $P_2 = 0.225 \text{ MPa}$	Single-end fixed	$4.2\mathrm{K}$	20.6	212 (S) 318 (1.5S)

Table 5. Summary of Stress Analysis Results Under Different Working Conditions

512 trol system continuously monitors the cavity's operational 513 parameters through sensors, predicts the potential detuning, 514 and generates corresponding corrective signals. These sig-515 nals are then used to adjust the tuner, temperature control, or 516 power supply systems to immediately correct the frequency 517 shift[47]. In this way, feedforward control allows for proac-518 tive measures to maintain frequency stability before detuning 519 occurs. This control method is particularly suitable for CW 520 operation, which requires high stability, as it ensures the cav-521 ity's frequency remains stable without interfering with the ac-522 celeration process, thus preventing performance degradation 523 due to frequency shifts[48].

#### The analysis of stress under different loading conditions

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As mentioned in previous conclusions, a smaller helium 525 vessel diameter positively impacts the reduction of df/dp, thus the diameter of the helium vessel is designed to be as 527 small as possible. To avoid interference between the bellows and the bare cavity, considering the outer diameter of the bare cavity is 106 mm, the inner diameter of the helium vessel was set to 116 mm, with a wall thickness of 4 mm. The INFN blade tuner is selected, with the bellows positioned at the center of the helium vessel. While this layout compromises the strength of the helium tank due to the presence of the bellows, it eliminates the need for additional longitudinal clearance, allowing for effective space savings[49–51]. The cavity model with a helium tank and bellows is shown in Fig. 16. 538

The purpose of the stress analysis is to ensure that the me-539 chanical design of the superconducting cavity can withstand the most challenging conditions encountered during its fabrication, testing, and operation. A comprehensive stress analysis of the dressed cavity was performed under four distinct 551 are within the permissible safety thresholds [52], P<sub>1</sub> is inside loading conditions, including the effects of the helium ves-  $_{552}$  cavity,  $P_2$  is inside helium vessel and  $P_3$  is outside helium sel and gravity. The loading scenarios examined include leak 553 vessel, as summarized in Table 5, confirming that the cavity tests, vertical tests, horizontal testing at cryogenic tempera- 554 meets the stress safety criteria for operation. The stress distriture, and extreme situation. The allowable stress limits for 555 bution under vertical test is shown in Fig. 17. The maximum 548 each material at room temperature and cryogenic tempera- 556 stress in the cavity is much lower than the material's allow-549 tures are provided in Table 1. Based on the simulation re- 557 able stress, and the cavity meets the strength requirements for

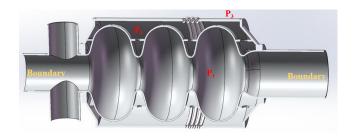


Fig. 16. Schematic of 3-cell cavity helium tank with bellows. Red area represents the loads of the stress analysis, and the yellow area represents the boundary conditions.

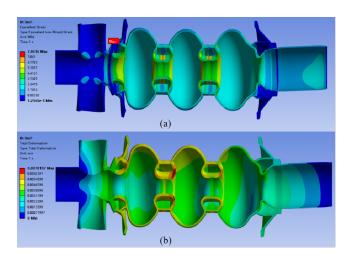


Fig. 17. (a) Stress and (b) deformation of the cavity under vertical test. Boundary and loading conditions depicted in Fig. 16 are used.

550 sults, the calculated stresses for all four loading conditions

558 stable operation under vertical test.

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#### VI. CONCLUSION

This study presents a comprehensive simulation and anal-560 ysis methodology for helium pressure-induced detuning in 561 562 high-current ERL superconducting cavities and structural optimizations to ensure long-term frequency and mechanical stability. A detailed investigation into the detuning mecha-565 nisms of the 1.3 GHz 3-cell injector cavity was conducted, 566 utilizing multiphysics simulations to analyze the factors in-567 fluencing helium pressure detuning. A predictive model for 568 helium pressure detuning was established. To ensure stable 569 operation of the superconducting cavity, the stiffening ring 570 position was optimized to 70 mm. This modification signifi- 594 cantly mitigated the impact of helium pressure detuning, re-572 ducing the frequency shift caused by helium bath pressure 596 and Development Program of China (2024YFA1612104), the variations from 45.3 Hz/mbar to 33.4 Hz/mbar, with a peak 597 CAS Project for Young Scientists in Basic Research (YSBR-<sub>574</sub> detuning of 10 Hz. Additionally, this design avoids the tun-575 ing challenges associated with the excessively large stiffening 599 nese Academy of Sciences, Shanghai Branch (JCYJ-SHFY-576 ring radius, achieving a tuning range of 500 kHz with a tuning 600 2021-010). 577 force of less than 15 kN, which significantly improves opera-578 tional feasibility. Furthermore, critical structural components 579 of the cavity were optimized, and their performance was rig-580 orously evaluated. The optimized design demonstrated excel-

lent results in stiffness, strength, and modal analysis, ensuring 582 robust structural stability.

In summary, this paper provides a thorough analysis 584 and design optimization of the mechanical structure of the 585 1.3 GHz 3-cell high-current superconducting cavity, signifi-586 cantly enhancing its frequency stability and mechanical performance. These advancements ensure its suitability for continuous-wave, high- $\beta$ , and high-current operation in ERL applications. The findings offer a valuable reference for the 590 mechanical design of superconducting cavities, contributing 591 to the development of more efficient and stable accelerator 592 systems.

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